

# COMPUTATION SHEET

**AECOM**

PROJECT NO. ....

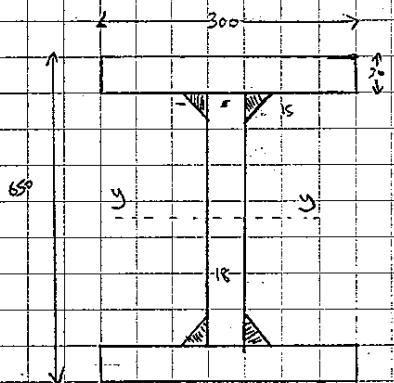
PAGE ..... OF .....

MADE BY ..... DATE .....

CHECKED BY ..... DATE .....

A7/11-12 51

SUBJECT CV 3202 Steel Design A



$$\begin{aligned}
 h &= 650 \text{ mm} \\
 b &= 300 \text{ mm} \\
 t_f &= 20 \text{ mm} \\
 h_w &= 650 - 20 - 20 = 610 \text{ mm} & h_w = h - 2t_f \\
 t_w &= 18 \text{ mm} \\
 s &= 15 \text{ mm}
 \end{aligned}$$

ki] Beam under bending

$$\epsilon = \sqrt{\frac{235}{275}} = 0.9244$$

outstand flanges:  $c = (b - t_w - 2s) / 2$   
 $= 126 \text{ mm}$

$$\frac{c}{t_f} = \frac{126}{20} = 6.3$$

Limit for class 1 flange =  $9\epsilon = 8.32 > 6.3$

$\therefore$  flange is class 1

web:  $c = h - 2t_f - 2s$   
 $= 580$

$$\frac{c}{t_w} = 32.22$$

Limit for class 1 web =  $12\epsilon = 66.6 > 32.2$

$\therefore$  web is class 1

$\therefore$  overall class fraction is class 1

ka] column under compression

$$\epsilon = 0.9244$$

$c = (b - t_w - 2s) / 2$   
 $= 126 \text{ mm}$

$$\frac{c}{t_f} = \frac{126}{20} = 6.3$$

Limit for class 1 flange =  $9\epsilon = 8.32 > 6.3$

$\therefore$  flange is class 1

web:  $c = h - 2t_f - 2s$   
 $= 580$

$$\frac{c}{t_w} = 32.22$$

Limit for class 2 web =  $38\epsilon = 35 > 32.22$

$\therefore$  web is class 2

$\therefore$  overall class fraction is class 2

PROJECT NO. ....

PAGE ..... OF .....

MADE BY ..... DATE .....

CHECKED BY ..... DATE .....

SUBJECT .....

1b

Plastic Shear of Welded I Section.

$$A_v = \eta \Sigma (h_w t_w) = 1$$

$$= 610 \times 18$$

$$= 10980 \text{ mm}^2$$

$$V_{pl,rd} = \frac{A_v \left( \frac{f_y}{\sqrt{3}} \right)}{\gamma_{mo}}$$

$$= 1743 \text{ kN}$$

Bending Resistance:

$$M_{pl,y,rd} = \frac{W_{pl,y} \cdot f_y}{\gamma_{mo}} = \frac{31871 \times 10^3 \times 275}{1} = 8764 \text{ kNm}$$

$$W_{pl,y} = (b \times t_f) \times (h - t_f) + b \times \left( \frac{h}{2} - t_f \right)^2$$

$$= (300 \times 20) \times (650 - 20) + 300 \left( \frac{650}{2} - 20 \right)^2$$

$$= 31871 \times 10^3 \text{ mm}^3$$

1c

Cross section compression resistance =>

$$N_{c,rd} = \frac{A_g}{\gamma_{mo}}$$

$$= \frac{(300 \times 20 \times 2 + 610 \times 18) \times 275}{1}$$

$$= 6320 \text{ kN}$$

Member buckling resistance in compression

$$I = \frac{1}{12} b h^3 - \left( \frac{1}{12} b h^3 \right)_{small} = 6.86 \times 10^9 - 5.33 \times 10^9 = 1.526 \times 10^6 \text{ mm}^4$$

$$N_{cr} = \frac{\pi^2 E I}{L_{eff}^2} = \frac{\pi^2 \times 210000 \times 1.526 \times 10^6}{5000^2}$$

$$= 126512 \text{ kN}$$

$$\bar{\lambda} = \sqrt{\frac{A_g}{N_{cr}}} = \sqrt{\frac{22980 \times 275}{126512 \times 10^3}} = 0.2234$$

$$\phi = 0.5 \left( 1 + 0.34 (0.2234 - 0.2) + 0.2234^2 \right)$$

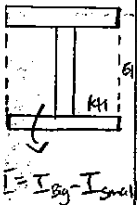
$$= 0.529$$

$$\chi = \frac{1}{\phi + \sqrt{\phi^2 - \lambda^2}} = 0.9916$$

$$N_{b,rd} = \chi \frac{A_g}{\gamma_{mi}}$$

$$= 0.9916 \times \frac{22980 \times 275}{1}$$

$$= 6267 \text{ kN}$$



# COMPUTATION SHEET

**AECOM**

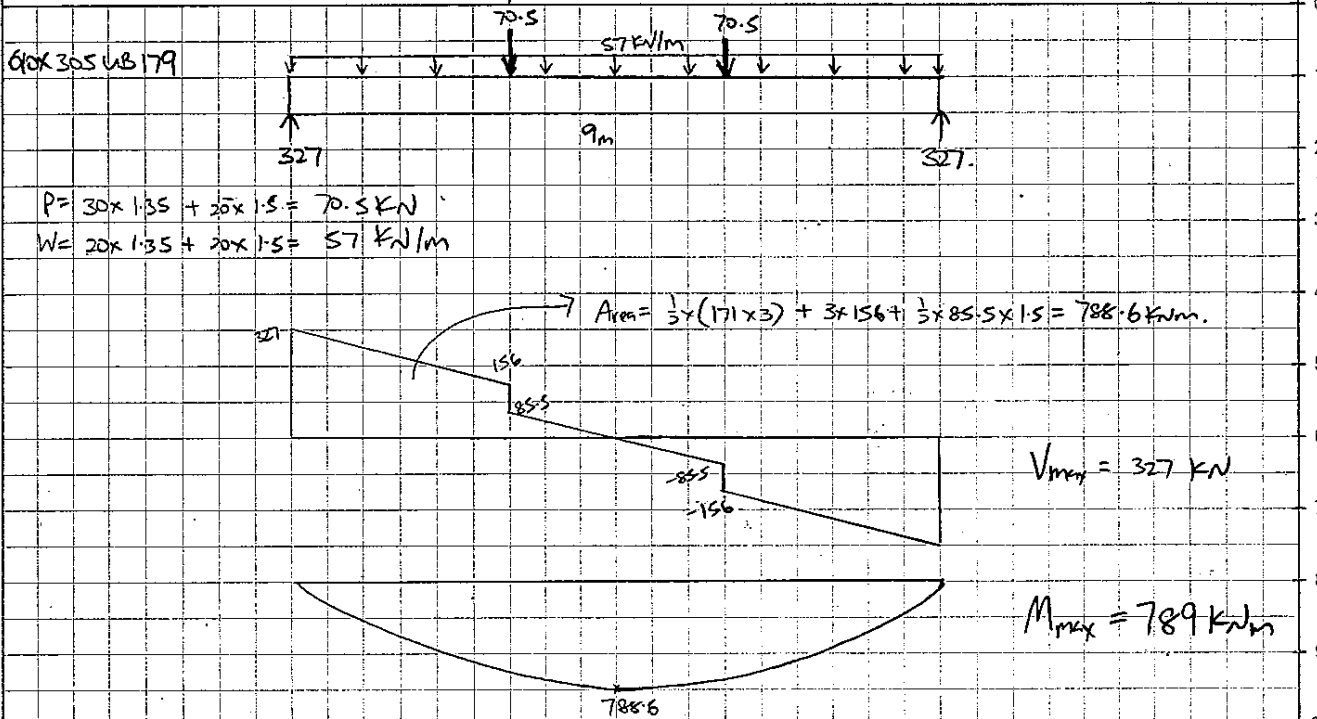
PROJECT NO. ....

PAGE ..... OF .....

MADE BY ..... DATE .....

CHECKED BY ..... DATE .....

SUBJECT .....



Shear check:

$$\begin{aligned}
 A_v &= A - 2b t_f + (t_w + 2t_f) t_w \geq \gamma_h w t_w \\
 &= 22800 - 2(307.1)(23.6) + (14.1 + 2 \times 16.5) \times 23.6 \geq 1 \times 573 \times 14.1 \\
 &= 9416 \text{ mm}^2 \geq 8079 \text{ mm}^2
 \end{aligned}$$

$$V_{\text{pl,red}} = \frac{A_v (f_y / \sqrt{3})}{\gamma_{mo}} = \frac{9416 \times (\frac{275}{\sqrt{3}})}{1.0} = 1495 \text{ kN} > V_{\text{max}} \Rightarrow \text{OK!}$$

High or low shear check:  $0.5 V_{\text{pl,red}} > V_{\text{max}}$   
 $748 \text{ kN} > V_{\text{max}} \rightarrow$  no reduction of moment resistance is required.

Bending Resistance:

$$M_{c,y,rd} = \frac{W_{pl,y} f_y}{\gamma_{mo}} = \frac{7486 \times 10^3 \times 275}{1} = 2059 \text{ kNm} > M_{\text{max}} \Rightarrow \text{OK!}$$

$A = 22800 \text{ mm}^2$   
 $r = 60.2 \text{ mm}$   
 $b = 307.1$   
 $t_f = 23.6$   
 $t_w = 14.1$   
 $r = 16.5$

# COMPUTATION SHEET

**AECOM**

PROJECT NO. ....

PAGE ..... OF .....

MADE BY ..... DATE .....

CHECKED BY ..... DATE .....

SUBJECT .....

LTB check:

$$M_{cr} = C_1 \sqrt{\frac{I_w}{I_z}} \sqrt{\frac{I_w}{I_z} + \frac{L^2 G J}{\pi^2 E I_z}}$$

$$= 1.77 \frac{\pi^2 \times 210000 \times 11407 \times 10^4}{9000^2} \sqrt{\frac{10.15 \times 100^6}{11407 \times 10^4} + \frac{9000^2 \times 81000 \times 34.1 \times 10^4}{\pi^2 \times 210000 \times 11407 \times 10^4}}$$

$$= 5167195 \times \sqrt{88965 + 94365}$$

$$= 2212 \text{ kNm}$$

$$\bar{\lambda}_{LT} = \sqrt{\frac{W_y f_y}{M_{cr}}}$$

$$= \sqrt{\frac{554710^3 \times 275}{2212 \times 10^6}} = 0.83$$

$$\phi_{LT} = 0.5 \left[ 1 + 0.21 (0.83 - 0.2) + 0.83^2 \right]$$

$$= 0.9106$$

$$\chi_{LT} = \frac{1}{0.9106 + \sqrt{0.9106^2 - 0.83^2}} = 0.778$$

$$M_{b,rd} = \chi_{LT} \frac{W_y f_y}{\gamma_{m1}}$$

$$= 0.778 \times \frac{554710^3 \times 275}{1}$$

$$= 1187 \text{ kNm} > M_{max}$$

Deflection:

Limiting Deflection:  $\frac{L}{360} = \frac{9000}{360} = 25 \text{ mm}$

$$\text{Total Deflection} = \frac{23 PL^3}{648 EI} \text{ (double point load)} + \frac{5}{384} \frac{WL^4}{EI} \text{ (UDL)}$$

$$= \frac{23 \times 70.5 \times 10^3 \times 9000^3}{648 \times 210000 \times 153025 \times 10^4} + \frac{5}{384} \times \frac{57 \times 9000^4}{210000 \times 153025 \times 10^4}$$

$$= 5.7 \text{ mm} + 15.15$$

$$= 20.9 \text{ mm} < 25 \text{ mm} \rightarrow \text{OK!}$$

# COMPUTATION SHEET

AECOM

PROJECT NO. ....

PAGE ..... OF .....

MADE BY ..... DATE .....

CHECKED BY ..... DATE .....

SUBJECT .....

3]

254 x 254 UC89 S 275

$$\text{Nominal moment about z-z Axis} = (150 - 100) \times \frac{b}{2} = 50 \times \frac{256.3}{2} = 6.4 \text{ kNm.}$$

$$\text{Nominal moment about y-y Axis} = 175 \times \frac{h}{2} = 175 \times \frac{260.3}{2} = 22.8 \text{ kNm.}$$

$$\text{Stiffness ratio} = \frac{4.8}{3.5} = 1.37 < 1.5 \rightarrow \text{moment distribute equally}$$

$$\Rightarrow \text{Design moment } M_{\text{red}} = \frac{22.8}{2} = 11.4 \text{ kNm.}$$

Flexural

Buckling about minor axis (z-z):

$$N_{\text{cr}} = \frac{\pi^2 EI}{L_{\text{cr}}^2} = \frac{\pi^2 \times 210000 \times 4860 \times 10^4}{4800^2} = 4372 \text{ kN}$$

$$\lambda_z = \sqrt{\frac{11500 \times 275}{4372 \times 10^3}} = 0.843$$

$$\phi_z = 0.5 \left[ 1 + 0.4 \lambda (0.843 - 0.2) + 0.843^2 \right]$$

$$= 1.013$$

$$\chi_z = \frac{1}{1.013 + \sqrt{1.013^2 - 0.843^2}}$$

$$= 0.635$$

$$N_{\text{brd}} = 0.635 \times 1500 \times 275 = 1974 \text{ kN} > N_{\text{max}} = 425 \text{ kN.}$$

LTB about major axis (y-y)

$$M_{\text{cr}} = 1.77 \times \frac{\pi^2 \times 210000 \times 4860 \times 10^4}{4800^2} \sqrt{\frac{0.717 \times 100^6}{4860 \times 10^4} + \frac{4800^2 \times 81000 \times 10^4 \times 10^4}{\pi^2 \times 210000 \times 4860 \times 10^4}}$$

$$= 7738310 \sqrt{4753 + 18898} = 1400 \text{ kNm.}$$

$$\lambda_{\text{LT}} = \sqrt{\frac{W_{\text{pl,y}}}{M_{\text{cr}}}} = \left[ \frac{1200 \times 10^3 \times 275}{1400 \times 10^6} \right]^{0.5} = 0.486$$

$$\phi_{\text{LT}} = 0.5 \left[ 1 + 0.2 \lambda (0.486 - 0.2) + 0.486^2 \right]$$

$$= 0.648$$

$$\chi_{\text{LT}} = \frac{1}{0.648 + \sqrt{0.648^2 - 0.486^2}} = 0.9286$$

$$M_{\text{brd}} = \chi_{\text{LT}} \frac{W_{\text{pl,y}}}{\gamma_{\text{mi}}} = 0.9286 \frac{1200 \times 10^3 \times 275}{1} = 311.6 \text{ kNm} > M_{\text{max}} = 11.4 \text{ kNm}$$

PROJECT NO. ....

PAGE ..... OF .....

MADE BY ..... DATE .....

CHECKED BY ..... DATE .....

SUBJECT .....

3c] 
$$\text{Max no of stories} = \frac{311.6}{11.4} = 27$$

4a] Shear: M20 class 8.8 x 3 bolts parallel to force.

Shear resistance of bolts =  $94.1 \text{ kN} \times 3 = 282.3 \text{ kN} > V_{max} = 175 \text{ kN} \rightarrow \text{OK!}$

Adequacy of 2x 6mm fillet welds (longitudinal resistance) =  $0.94 \times 180 \text{ mm} \times 2$   
 =  $338.4 \text{ kN} > V_{max} = 175 \text{ kN} \rightarrow \text{OK!}$

4b] plan shear of fin plate: 
$$V_{p,rd} = \frac{A_v \left( \frac{f_y}{\sqrt{3}} \right)}{\gamma_{mo}} \times 2 \rightarrow A_v = A - A_h$$
  

$$= (180 \times 9) - (9 \times 20) = 1440 \text{ mm}^2$$
  

$$= \frac{1440 \left( \frac{275}{\sqrt{3}} \right)}{1} \times 2 \text{ plates}$$
  

$$= 457.3 \text{ kN} > V_{max} = 175 \text{ kN}.$$

Block tear resistance of fin plate: For bolt group subject to eccentric loading: 
$$V_{eff,rd} = 0.5 \frac{f_u A_{nt}}{\gamma_{m2}} + \frac{1}{\sqrt{3}} \left( \frac{f_y A_{nv}}{\gamma_{m0}} \right) \times 2 \text{ plates}$$
  

$$f_u = 430 \text{ N/mm}^2$$
  

$$A_{nt} = \left[ 60 - \left( \frac{20}{2} \right) \right] \times 9 = 450 \text{ mm}^2$$
  

$$A_{nv} = [4 + 5 + 5 + 5 + 5] \times 9 = 810 \text{ mm}^2$$
  

$$= 0.5 \frac{430(450)}{1} + \frac{1}{\sqrt{3}} \left[ \frac{275(810)}{1} \right] \times 2$$
  

$$= 322.1 \text{ kN} > V_{max} = 175 \text{ kN}$$

4c] Weakest link of this typical connection are the 3 M20 class 8.8 bolts with shear resistance of 282.3 kN

THE END

\* All the BEST for the EXAMS \*

Done By: Zhihui